Treating of "Automated Valve Assemblies" as "Engineered Products"

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What does the implementation of Recommended Practice S2812-X-19 or ISO/TC 153 N 425, ISO/NP 5115 in practice mean for the various groups involved such as valve and actuator manufacturers, integrators, and end-users of automated valve assemblies?

In June 2019, the "Recommended Practice S2812-X-19" (hereinafter referred to as RP) was published by the WIB at the Valve World Americas Conference in Houston [1]. The RP is entitled "Actuated Valve Assembly - A Recommended Practice for Part turn Automated On-Off valves" and addresses the issue (including the definition of the general conditions) of sizing, selection, and the mechanical integrity of automated valve assemblies widely used in many industries.

NEW MODULE FOR CONVAL

Since F.I.R.S.T. has comprehensively supported the development of the RP both in principle and with functional prototypes, it is a matter of course that a module will soon be available in CONVAL^{*} that fully maps the RP. Recognizing the importance of the issue for the worldwide community, the ISO has taken up the task of developing an international ISO standard with RP as a basis [2]. At this point, the ISO even goes one step further than the RP in considering electric actuators, as the RP is limited to pneumatic (and in a broader context hydraulic) actuators.

I have already published a paper on this topic in Industrial Valves, so I will not go into all the details of the RP here but only give an overview. The focus will be on the requirements for the implementation of the RP in practice, from the various perspectives. The viewpoint of the manufacturers of the valves, the actuators, possible integrators as well as that of the end-users.

THE CORRECT DESIGN OF ACTUATORS

Recently I gave two webinars on this topic at the virtual "Forum Industriearmaturen" and during the introduction, I made a small poll among the participants. Each one of the webinar participants confirmed that they had cases of valves in their practice that did not open or close when required or even sheared off the stem. For me a clear indication of the importance of the topic for the industrial valve user community.

If one were to summarise the objectives of the RPs in a few sentences, it could be described as follows:

- If you automate rotary valves as shut-off valves, you have to find the right actuator for the required torque, which is derived from the valve and application data.
- On the one hand, this actuator should have enough torque and an adequate safety margin to operate the valve safely under all expected conditions.
- On the other hand, it should not be oversized, and not just for cost reasons. It must not be designed in such a way that the torque delivered may damage the valve, seat, or stem.
- The design should take into account all relevant data and conditions of the equipment used and the application itself.
- The selection made should be documented comprehensively, completely, and uniformly so that it can be traced at any time.

9		Environmental conditions			corros	sive			End user
Applicati	ion								
		Requested acting mode			Spring	a return			End user
10	1	Valve fail safe action			Fail cl	ose			End user
11		Travel time	open/close			10.0	5.	0 s	End user
12		Response time	open/close			20,0	8,	0 s	End user
13		Air supply pressure	min/max			4,0	10,	0 bar(g)	End user
Process								1	
14		Medium			hvdro	carbon +	- tar		End user
15		State / phase							End user
16		Mass flow rate					100.000	0 ka/h	End user
17		Volume flow rate					125	$0 m^3/h$	Enduser
18		Density					900	0 ka/m ³	End user
19	1.3	Long standstill time					11.	0 month(s)	End user (1)
20	1.4	Non-clean service			Yes				End user (1)
	.,.	Slurries			No				End user (1)
		Chrystallizing or polymerizing media							End user (1)
									End user (1)
		Sticky, non-lubricating light	uid		No				End user (1)
		Non-lubricating dry gas			No				End user (1)
23		Fluid operation temp.	min/max	t1,min		10,0	40,	0 °C	End user (1)
24		Max. shut off pressure (val	ve closed)	Δp			80,	0 bar	End user
25		Design pressure					28.98	7 bar(g)	End user
Valve	- 7								
27		Valve manufacturer			Sern				Enduser
28		Valve series			Full n	ort class	150		Enduser
29	_	Valve type			Ball va	alve	150		Enduser
30		Valve design			Trunn	ion mou	nted		Enduser
31		Port type			Full p	ort	incea -		Enduser
32		Flow direction			i un p				Enduser
33		Seat sealing type			Soft s	eated			Enduser
34		Seat material							Enduser
35		Seating method			Positi	on seater	1		Enduser
36		Pressure rating			Class	150			Enduser
37		Tightness rate/class							Enduser
38		Selected valve size			10"				End user
40	1.4	Break to open torque	net/ODCF corr.	BTO		1.105.0	1.547	0 Nm	Valve mfr
41		Breakaway angle		θ			10	0 •	Valve mfr
42	1.4	Run to open torque	net/ODCF corr.	RTO		635.0	889	0 Nm	Valve mfr
43	1.4	End to open torque	net/ODCF corr.	ETO		695.0	973.	0 Nm	Valve mfr
44	1,82	Break to close torque	net/ODCF corr.	BTC	()	695,0	1.264	9 Nm	Valve mfr
45	1.4	Run to close torque	net/ODCF corr.	RTC	()	635.0	6 889	0 Nm	Valve mfr
47	14	End to close torque	net/ODCF corr	ETC		885.0	1,239	0 Nm	Valve mfr
48	.,-	Max, allowable stem torou	e	MAST		000,0	5.466	0 Nm	Valve mfr
49		Max, flange torque	-	f.max			4.000	0 Nm	Valve mfr
50		Stem / top works dimensio	ons provided	.,	4.000,0 Nm			Valve mfr	
Mountin	na kit								
51	ig kit	Material							Mounting Vit mf
52		Max allowable coupling to		MAST				Nm	Mounting Kit mf
54		MK mechanical integrity of	hecked and docur	mented	No			NIII	Mounting Kit mf
54		MK mechanical integrity cl	hecked and docur	mented	No			INIT	Mounting Kit r

Figure 1: "Automated Valve Data Sheet", extract

When you look at the RP it quickly becomes clear that the approach described makes perfect sense by itself, but immediately raises many more questions and demands on the groups involved. I have already outlined these in the DIAM 2019 lecture program [3]. It should not be forgotten that RP was initially conceived from the perspective and driven by the needs of endusers in the process industry. However, to implement it, a profound acceptance, transparency, and openness on the part of the manufacturers of actuators and especially valves are required.

AUTOMATED VALVES AND ENGINEERED PRODUCTS

But first to the question "Why all this?". In short, because today automated valves are not treated as "Engineered

Maximum Allowable Stem Torque (MAST)

Table B - Analytical calculations based on Roark formulas

Stem	Unit	316/316L A479 S31600/ S31603	17-4PH A564 H1150D S17400	Alloy C22 B574 N06022	Alloy 20 B473 N08020	Monel 400 A164 N04400	Duplex A479 S31803	Super Duplex A479 S32750	254 SMO A479 S31254	Titanium Gr.2 B348 R50400	Inconel 718 B637 N07718
16"	Nm	9.90	29.02	12.98	8.20	8.20	15.37	17.59	10.41	9.39	35.34
72	inch*lbs	87.64	256.89	114.84	72.53	72.53	136.00	155.64	92.18	83.11	312.80
	Nm	22.63	66.34	29.66	18.73	18.73	35.12	40.20	23.80	21.46	80.78
	inch*lbs	200.33	587.17	262.50	165.79	165.79	310.86	355.76	210.69	189.97	714.97
	Nm	39.61	116.10	51.90	32.78	32.78	61.46	70.34	41.66	37.56	141.37
1 72	inch*lbs	350.58	1027.55	459.38	290.13	290.13	544.00	622.57	368.71	332.44	1251.19
214"	Nm	137.22	402.20	179.80	113.56	113.56	212.93	243.68	144.32	130.12	489.73
272	inch*lbs	1214.50	3559.73	1591.41	1005.10	1005.10	1884.56	2156.78	1277.31	1151.68	4334.49
2"	Nm	230.59	675.85	302.15	190.83	190.83	357.80	409.49	242.51	218.66	822.95
3	inch*lbs	2040.85	5981.81	2674.22	1688.98	1688.98	3166.84	3624.27	2146.41	1935.29	7283.74
	Nm	202.29	592.93	265.07	167.41	167.41	313.90	359.24	212.76	191.83	721.98
3 00	inch*lbs	1790.44	5247.85	2346.10	1481.75	1481.75	2778.27	3179.58	1883.05	1697.83	6390.03
¢"	Nm	792.20	2321.95	1038.05	655.61	655.61	1229.27	1406.83	833.17	751.22	2827.32
•	inch*lbs	7011.52	20551.01	9187.51	5802.64	5802.64	10879.95	12451.49	7374.19	6648.86	25023.88

Figure 2: MAST (Maximum Allowable Stem Torque)

Product". Safety, application, and possible further factors remain unclear and the procurement of valve, coupling, and the actuator is often in different hands. As a result, they are not considered as assemblies and responsibilities for the final delivered solution often remain unclear. The RP clearly defines the responsibilities for the required data and keeps them in the documentation, the "Automated Valve Data Sheet". There, the right-hand column

		3 BAR ASYMMETRISCH							
Modellnummer	Maximales Drehmoment Nm	Abtriebsdrehmoment (Nm)							
		BTO	RTO	ETO	BTC	RTC	ETC		
XXXXXXX	9.000	2.327	908	1.289	1.229	895	2.528		
XXXXXXX	9.000	3.304	1.290	1.830	1.754	1.278	3.607		
XXXXXXX	9.000	4.730	1.847	2.620	2.513	1.830	5.167		
XXXXXXX	9.000	6.269	2.448	3.472	3.340	2.433	6.867		
XXXXXXX	9.000	8.030	3.135	4.448	4.286	3.122	8.812		
xxxxxxx	18.000	5.637	2.229	3.122	2.995	2.238	6.157		
XXXXXXX	18.000	7.471	2.955	4.138	3.980	2.974	8.183		
xxxxxxx	18.000	9.570	3.785	5.300	5.107	3.816	10.502		
XXXXXXX	18.000	11.896	4.705	6.589	6.357	4.750	13.071		
XXXXXXX	18.000	14.441	5.665	7.921	7.664	5.625	15.794		
XXXXXXX	32.000	9.808	3.830	5.433	5.212	3.797	10.715		
XXXXXXX	32.000	12.564	4.906	6.960	6.677	4.864	13.728		
XXXXXXX	32.000	15.618	6.098	8.651	8.319	6.060	17.102		
xxxxxxx	32.000	19.069	7.446	10.563	10.173	7.410	20.914		
XXXXXXX	32.000	22.877	8.933	12.672	12.219	8.901	25.121		
XXXXXXX	32.000	27.046	10.561	14.982	14.459	10.533	29.725		

Figure 3: Different torques

clearly shows who is responsible for the data (**Figure 1**). Nothing is surprising on the end-user side, the data can all be obtained from the process engineering department and the valve manufacturer. However, if you want to choose a suitable actuator, you will need a lot more information about the valve, the mounting kit, and the potential actuator.

Ideally, as implemented in CONVAL^{*}, the requirements of the process are specified, supplemented by information on factors (media properties, long standstill times, etc.) that could further influence the torque requirement, a valve is selected from the database and, after entering a safety factor for the design, the appropriate actuator can be selected.

This ideal case, however, requires that valve and actuator manufacturers have determined and published the required key parameters of their devices and that these are also stored in the database. Of course, manual input is always possible if the data is not available in the database. On the actuator side, this is standard, usually the data on torques, flange design according to ISO 5211, etc. are published in full [4]. Even if these have not yet been captured in the database, the entry is quickly completed with just a few values, taken directly from the documentation of the actuator manufacturer.

With the valves, the situation is different for a variety of reasons. Things like the MAST (Maximum Allowable Stem Torque) can still be found, depending on the stem

material (example Figure 2), as they are more or less easy to calculate by the manufacturer [5]. The flange class according to ISO 5211 is also no problem. However, RP still expects, for the calculation of the required torque of the valve, six different torques (as shown in Figure 3 as an example) matching the differential pressure. Furthermore, a breakaway angle as well as correction factors for the application are required.

PENT-UP DEMAND FOR TECHNICAL DATA

Sometimes, however, the maximum torque is the only information available. There are rules of thumb (for each valve style) like e.g. BTO, ETC = 100%, RTO, RTC=40%-50%, ETO, BTC=70%,

Corrections factors - special application

Emergency shut-down (ESD) service	1.8
IEC 61508 SIL complaint installation	1.8
Cryogenic applications (below -60 $^\circ$ C)	1.5
Valves operated less than once a day	1.5
Control valves	1.5

Corrections factors - Media

Gas, dirty (natural gas)	1.5
Gas , dry	1.3
Chlorine	1.5
Viscous slurry (cp>100)	2.0
Oil, thermal oil, lubricant	0.8

Figure 4: Various correction factors

but these should be used with caution and do not conform to the RP or the upcoming ISO standard.

In practice, however, there are unfortunately hardly any publicly available and fully documented torques, so that there seems to be a considerable backlog demand on the part of the valve manufacturers.

The situation is similar for the correction factors for the application conditions (defined as ODCFs in the RP). Some manufacturers specify factors for different situations (example Figure 4), but these do not necessarily corres-

Korrekturfaktoren für besondere Anforde	erungen (ODCF)		×
Hier können Sie die Standardwerte für Die Standardwerte werden verwendet, aktuellen CONVAL ß-Version werden die	die Korrekturfa wenn für das g ese Werte noch	iktoren für besondere Anwer iewählte Ventil keine ODCF-W n nicht mit der Berechnung g	ndungen (OD) /erte vorliege espeichert.	CF) festlegen. n. In der
Langzeitstillstand				
1. Faktor Langzeitstillstand	1,3 ~	1. Schwelle	1 .	Monat(e)
2. Faktor Langzeitstillstand	1,5 ~	2. Schwelle	12 .	Monat(e)
Medium				
Faktor min. Temperatur	1,3 ~	Min.Temperatur	-150	- °C
Faktor max. Temperatur	1,3 ~	Max. Temperatur	500	~ °C
Nicht reiner Betrieb	1,4 ~	Nicht schmierendes Trock	engas	1,3 ~
Klebrige, nicht schmierende Flüssigkeit	1,4 ~	Suspensionen / Schlämme	e	1,5 ~
Schmierendes Medium	0,8 ~	Kristallisation oder Polymo	erisation	1,6 ~
Berechnungsmethode (Medium)	Summe ~			
Berechnungsmethode (kombiniert)	Produkt ~		Erweiterte Ei	nstellungen
		Standard	ОК	Abbrechen



ODCF - Zuordnung zu Drehmomenten

Hier können Sie wählen, welche Drehmomente von den einzelnen Faktoren beeinflusst werden. Dabei wird zwischen nur bei Feder schließt (FC), nur bei Feder öffnet (FO) oder unabhängig von der Betriebsart (FC/FO) unterschieden.

	BTO	RTO	ETO	BTC	RTC	ETC
🗏 Typ : Langer Stillstand						
Lange Stillstandszeit	FO	-	-	FC	-	-
Typ : Mediumeigenschaft						
Nicht reiner Betrieb	FC/FO	FC/FO	FC/FO	FC/FO	FC/FO	FC/FO
Schmierendes Medium	FC/FO	FC/FO	FC/FO	FC/FO	FC/FO	FC/FO
Suspensionen / Schlämme	FC/FO	FC/FO	FC/FO	FC/FO	FC/FO	FC/FO
Klebrige, nicht schmierende Flüssigkeit	FC/FO	FC/FO	FC/FO	FC/FO	FC/FO	FC/FO
Kristallisation oder Polymerisation	FC/FO	FC/FO	FC/FO	FC/FO	FC/FO	FC/FO
Nicht schmierendes Trockengas	FC/FO	FC/FO	FC/FO	FC/FO	FC/FO	FC/FO
Typ : Mediumtemperatur						
Min. temperature	FC/FO	FC/FO	FC/FO	FC/FO	FC/FO	FC/FO
Max. Temperatur	FC/FO	FC/FO	FC/FO	FC/FO	FC/FO	FC/FO
			Standar	d (ОК А	Abbrechen

Figure 6: Assignment to torques

Antriebsauswa	hl								
🔶 Favoriten	Empfehlung	g 🙀 Aktueller Antr	rieb Keir	n Filter 🛛	Kennlinie	en ODCF	-Einstellung	en	ŝ
Die Auswahl	ist auf geeigne	te Antriebe gefilt	ert (44)						
Ansicht gruppi	ert nach Hersteller	Kombinierte Ansich	t						
Hersteller	Baureihe	Modell	SI 🛆	Masse	Vol.	p,min	p,max	Тур	^
Bray	S98 SR	73E2-12-1 S	0,10	224,0	14,8	4,14	9,58	SY Symm.	
Rotork	GP SR	GP-065S-385A/C3	0,10			4,0	6.0	SY Symm.	
Bray	S98 SR	73E2-12-1 C	0,11	224,0	14,8	4,14	8,27	SY Canted	
Velan ABV	PS	PS1/S-A/150/X3	0,13			3,5	6,0	SY Symm.	
Limitorque	LPS SR	20A-335X-FX2-2	0,14	320,0	20,0	4,0	11,3 :	SY Symm.	
Ledeen	SR	S:s006-0032-10a	0,14			4,0	7,0 :	SY Symm.	
Bray	S98 SR	45E2-14-4 S	0,14	199,0	16,7	4,14	6,14	SY Symm.	
Pfeiffer	BR 31a - SRP	05000:5-6	0,15	198,0	40,0	5,0	8,0	R&P	
Air Torque	PT	PT800 S 11	0,15	198,0	40,0	5,0	8,0	R&P	
Air Torque	PT	PT800 S 10	0,15	192,0	40,0	4,5	8,0	R&P	
Pfeiffer	BR 31a - SRP	05000:5-5	0,15	192,0	40,0	4,5	8,0	R&P	
Ledeen	SR	S:s008-0033-2a	0,15			4,0	7,0 5	SY Symm.	
Pfeiffer	BR 31a - SRP	05000:4-5	0,16	187,0	40,0	4,0	8,0	R&P	
Air Torque	PT	PT800 S 09	0,16	187,0	40,0	4,0	8,0	R&P	
Limitorque	LPS SR	25A-300X-FX1-2	0,17	383,0	19,8	5,0	12,0 :	SY Symm.	
Limitorque	LPS SR	20A-360X-FX2-3	0.20	354.0	23.1	4.5	10.2	SY Symm.	~
Antrieb						Filter	r		
Sicherheitsfr	aktor Versorgu	nasluftdruck							
SCE 12		50	6.0		<i>(</i> -)	Gefo	orderte Betrie	ebsart	
551 1,2	p,min	5,0 p,max	0,0	bar	(g)	Pne	eumatisch		
						Fed	lerrücksteller	nd	
Sicherheitsrese	Drehmome	nte [Nm]				Fed	ler schließt		
	Ventil zu	Im Betrieb	/entil auf	Lost	orechwinkel				
Federmome	ente 43 %	32 %	92 %		20 %		Scotch-Yoke		~
			70.0/		22.04		. Adaptation of the second	ente filtere	
Luftmomer	49 %	24 %	/0 %		23 %		untarenmon	iente nitern	
	MAST	Flansch					lanschmom	ente berücksi	chtigen
Max. Mom	ente 31 %	6 %							
	-						_		
Hilfe						Ok	Abbre	chen Üb	ernehmen

Figure 7: Easy actuator selection

pond to the system of factors in the ODCFs and certainly do not define how to proceed when several factors come together (add, increase, multiply, etc.). As defined by the RP, CONVAL[®] allows each manufacturer of valves to configure these factors for each valve series and to define how they are handled individually. This applies to the factors and their application limits (see Figure 5) as well as to the question to which torque the factor should be applied (see Figure 6). For example, by default (RP recommendation) the factor for long standstill times for the safety function "spring closes" is only applied to the BTC, for "spring opens" only to the BTO. In the case of series-production valves (larger number of pieces/year), there is a chance to obtain this data from the manufacturer, but in the case of so-called "one of a kind", i.e. unique valves specially designed for the case, the effort and cost to determine the data are enormously high. In this case, the user can only make progress through intensive cooperation with the manufacturer.

RP CONFORMS ACTUATOR SELECTION IS POSSIBLE

The described situation makes it very clear how detailed the determination of the required torques for the safe operation of the valve is implemented in the RP and will soon be required by ISO. It is obvious which hurdles still have to be overcome before an end-user, or whoever has to find the right actuator for a valve, can simply follow the RP and comply with the ISO.



Figure 8: Graphical representation of the selected actuator

But once the hurdles have been overcome and the necessary data made available, these can, in combination with a software module such as in CONVAL^{*}, make design, selection, and documentation considerably easier, more reliable, and reproducible.

If all the data is available, a suitable actuator, automatically rated and sorted by a "Suitability Index" (KPI for the suitability of the actuator for the application) is suggested and the selection is clearly structured (**Figure 7**) and graphically supported (**Figure 8**).

It is clear that the path is well described and tools for implementation will soon be available.

Author



However, based on decades of practical experience in the field of control valves, I assume that it will still take considerable time and effort until device data is generally available and of good quality.

But that should not prevent us from embarking on this approach now.

LITERATURE

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